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> Proposal for Supplement 7 to the 06 series of amendments to UN Regulation No. 49 (Emissions of compression ignition and positive ignition (LPG and CNG) engines)

Submitted by the Working Party on Pollution and Energy*

The text reproduced below was adopted by the Working Party on Pollution and Energy (GRPE) at its eighty-third session (ECE/TRANS/WP.29/GRPE/83, para. 22). It is based on ECE/TRANS/WP.29/GRPE/2021/14, as amended by Annex X of the report. It is submitted to the World Forum for Harmonization of Vehicle Regulations (WP.29) and to the Administrative Committee (AC.1) for consideration at their November 2021 sessions.

^{*} In accordance with the programme of work of the Inland Transport Committee for 2021 as outlined in proposed programme budget for 2021 (A/75/6 (part V sect. 20) para 20.51), the World Forum will develop, harmonize and update UN Regulations in order to enhance the performance of vehicles. The present document is submitted in conformity with that mandate.





Annex 4

Paragraph 8.2., amend to read:

"8.2. NOx correction for humidity

As the NOx emission depends on ambient air conditions, the NOx concentration shall be corrected for humidity with the factors given in paragraph 8.2.1. or 8.2.2. The intake air humidity Ha may be derived from relative humidity measurement, dew point measurement, vapour pressure measurement or dry/wet bulb measurement using generally accepted equations.

For all humidity calculations (for example Ha, Hd) using generally accepted equations the saturation vapour pressure is required. For calculating the saturation vapour pressure which is in general a function of the temperature (at the humidity measurement point) the equation D.15 specified in Annex D to ISO Standard 8178-4:2020 should be used."

Paragraph 8.4.2.3., Equation (36), amend to read:

"...

The following equation shall be applied:

$$m_{gas} = u_{gas} \times \sum_{i=1}^{l=n} \left(c_{gas,i} \times q_{mew,i} \times \frac{1}{f} \right)$$
 in (g/test) (36)

Where:

..."

Paragraph 8.4.2.4., Equation (37), amend to read:

"...

The following equation shall be applied:

$$m_{gas} = \sum_{i=1}^{i=n} \left(u_{gas,i} \times c_{gas,i} \times q_{mew,i} \times \frac{1}{f} \right)$$
 in $\left(\frac{g}{test} \right)$ (37)

Where:

. . . "

Paragraph 8.5.1.4., Equation (54), amend to read:

"...

$$Q_{SSV} = \frac{A_0}{60} d_V^2 C_d p_p \sqrt{\left[\frac{1}{T} \left(r_p^{1.4286} - r_p^{1.7143}\right) \cdot \left(\frac{1}{1 - r_D^4 r_p^{1.4286}}\right)\right]}$$
 (54)

Where:

$$A_0$$
 is 0.005692 in SI units of $\left(\frac{m^3}{\min}\right)\left(\frac{K^{\frac{1}{2}}}{kPa}\right)\left(\frac{1}{mm^2}\right)$

 $d_{\rm V}$ is the diameter of the SSV throat, mm

. . . "

Paragraph 8.5.2.3.1., Equation (57), amend to read:

$$u_{gas} = \frac{M_{gas}}{M_d \times \left(1 - \frac{1}{D}\right) + M_e \times \left(\frac{1}{D}\right)} \times \frac{1}{1000}$$
 (57)

• • •

Paragraph 8.6.1., amend to read:

"...

Depending on the measurement system and calculation method used, the uncorrected emissions results shall be calculated with equations 36, 37, 56, 58 or 62, respectively. For calculation of the corrected emissions, $c_{\rm gas}$ in equations 36, 37, 56, 58 or 62, respectively, shall be replaced with $c_{\rm cor}$ of equation 66. If instantaneous concentration values $c_{\rm gas,i}$ are used in the respective equation, the corrected value shall also be applied as instantaneous value $c_{\rm cor,i}$. In equations 58 and 62, the correction shall be applied to both the measured and the background concentration.

...'

Paragraph 9.2., Table 7, amend to read:

"Table 7 Linearity requirements of instruments and measurement systems

Measurement system	χmin X (a1 - 1)+ a0∖	Slope a1	Standard error SEE	Coefficient of Determination r2
Engine speed	≤ 0.05 % max	0.98 - 1.02	≤ 2 % max	≥ 0.990
Engine torque	≤ 1 % max	0.98 - 1.02	≤ 2 % max	≥ 0.990
Fuel flow	≤ 1 % max	0.98 - 1.02	≤ 2 % max	≥ 0.990
Airflow	≤ 1 % max	0.98 - 1.02	≤ 2 % max	≥ 0.990
Exhaust gas flow	≤ 1 % max	0.98 - 1.02	≤ 2 % max	≥ 0.990
Diluent flow	≤ 1 % max	0.98 - 1.02	≤ 2 % max	≥ 0.990
Diluted exhaust gas flow	≤ 1 % max	0.98 - 1.02	≤ 2 % max	≥ 0.990
Sample flow	≤ 1 % max	0.98 - 1.02	≤ 2 % max	≥ 0.990
Gas analyzers	≤ 0.5 % max	0.99 - 1.01	≤ 1 % max	≥ 0.998
Gas dividers	≤ 0.5 % max	0.98 - 1.02	≤ 2 % max	≥ 0.990
Temperatures	≤ 1 % max	0.99 - 1.01	≤ 1 % max	≥ 0.998
Pressures	≤ 1 % max	0.99 - 1.01	≤ 1 % max	≥ 0.998
PM balance	≤ 1 % max	0.99 - 1.01	≤ 1 % max	≥ 0.998
Humidity measurement device	≤ 2 % max.	0.98 – 1.02	≤ 2 %	≥ 0.95

Paragraph 9.3.3.1., amend to read:

"9.3.3.1. Pure gas

. . .

Hydrogen mixture (FID burner fuel) $(40 \pm 1 \text{ per cent hydrogen, balance helium or alternatively nitrogen)}$ (Contamination $\leq 1 \text{ ppm C1}, \leq 400 \text{ ppm CO2}$)"

Paragraph 9.3.6.2., amend to read:

"9.3.6.2. Calibration

The CLD and the HCLD shall be calibrated in the most common operating range following the manufacturer's specifications using zero and span gas (the NO content of which shall amount to about 80 per cent of the operating range and the NO2 concentration of the gas mixture to less than 5 per cent of the NO concentration). With the ozonator deactivated, the NOx analyzer shall be in the

NO mode so that the span gas does not pass through the converter. The indicated concentration has to be recorded."

Paragraph 9.3.6.8., amend to read:

"9.3.6.8. NOx mode

Keeping NOx mode with the ozonator deactivated, the flow of oxygen or synthetic air shall be shut off. The NOx reading of the analyzer shall not deviate by more than ± 5 per cent from the value measured according to paragraph 9.3.6.2. (the analyzer is in the NOx mode)."

Paragraph 9.5.4.1., amend to read:

"9.5.4.1. Data analysis

...

$$C_d = \frac{Q_{SSv}}{\frac{A_0}{60} \times d_V^2 \times p_p \times \sqrt{\left[\frac{1}{T} \times (r_p^{1.4286} - r_p^{1.7143}) \times \left(\frac{1}{1 - r_D^4 \times r_p^{1.4286}}\right)\right]}}$$
(89)

Where:

 $Q_{\rm SSV}$ is the airflow rate at standard conditions (101.3 kPa, 273 K), m³/s

T is the temperature at the venturi inlet, K

 $d_{\rm V}$ is the diameter of the SSV throat, mm

. . .

$$Re = A_1 \times 60 \times \frac{Q_{SSV}}{d_V \times \mu} \tag{90}$$

With

$$\mu = \frac{b \times T^{1.5}}{S + T} \tag{91}$$

Where:

A₁ is 27.43831 in SI units of
$$\left(\frac{kg}{m^3}\right)\left(\frac{min}{s}\right)\left(\frac{mm}{m}\right)$$

 $Q_{\rm SSV}$ is the airflow *rate* at standard conditions (101.3 kPa, 273 K), m³/s

 $d_{\rm V}$ is the *diameter* of the SSV throat, mm

..."

Annex 4, Appendix 2

Paragraph A.2.1.3., amend to read:

"A.2.1.3. Components of Figures 9 and 10

EP Exhaust pipe

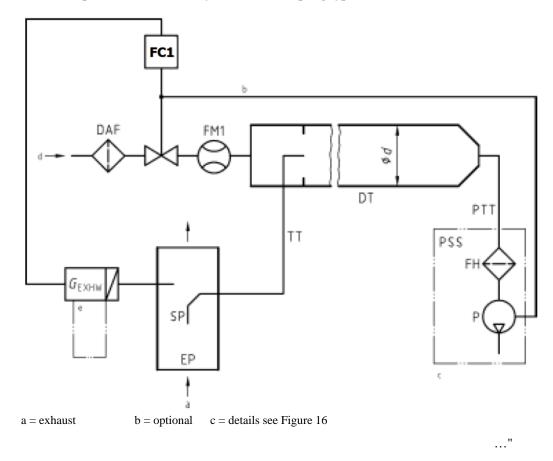
SP1 Raw exhaust gas sampling probe (Figure 9 only)

..."

Paragraph A.2.2.1., amend to read:

"...

Figure 12 Scheme of partial flow dilution system (total sampling type)



Paragraph A.2.2.5., amend to read:

"...

For a partial flow dilution system, a sample of the diluted exhaust gas is taken from the dilution tunnel DT through the particulate sampling probe PSP and the particulate transfer tube PTT by means of the sampling pump P, as shown in Figure 16. The sample is passed through the filter holder(s) FH that contain the particulate sampling filters. The sample flow rate is controlled by the flow controller FC2.

For of full flow dilution system, a double dilution particulate sampling system shall be used, as shown in Figure 17. A sample of the diluted exhaust gas is transferred from the dilution tunnel DT through the particulate sampling probe PSP and the particulate transfer tube PTT to the secondary dilution tunnel SDT, where it is diluted once more. The sample is then passed through the filter holder(s) FH that contain the particulate sampling filters. The diluent flow rate is usually constant whereas the sample flow rate is controlled by the flow controller FC2. If electronic flow compensation EFC (see Figure 15) is used, the total diluted exhaust gas flow is used as command signal for FC2.

..."